
Introduction to Computational Fluid Dynamics

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Graduate course

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1 Principles and equations of Fluid Mechanics

1.1 Continuous media

- The continuum hypothesis.
- What is a material point?
- The velocity.

1.2 Cartesian vectors and tensors

We assume $\{x_1, x_2, x_3\}$ to be Cartesian coordinates, with

$$\check{e}^{(1)}, \quad \check{e}^{(2)}, \quad \check{e}^{(3)} \tag{1.1}$$

the Cartesian basis of vectors.

Vector field:

$$\mathbf{u}(\mathbf{x}, t) = \sum_i u_i(\mathbf{x}, t) \check{e}^{(i)} \tag{1.2}$$

Gradient:

$$\nabla \varphi = \sum_i \frac{\partial \varphi}{\partial x_i} \check{e}^{(i)} = \varphi_{,i} \check{e}^{(i)} \tag{1.3}$$

$$\underline{\nabla} \varphi = (\varphi_{,1}, \varphi_{,2}, \varphi_{,3})^T \tag{1.4}$$

Divergence:

$$\nabla \cdot \mathbf{u} = \sum_i \frac{\partial u_i}{\partial x_i} = u_{i,i} \tag{1.5}$$

Tensor product of two vectors:

$$\mathbf{u} \otimes \mathbf{v} = \sum_{i,j} u_i v_j \check{e}^{(i)} \otimes \check{e}^{(j)} \tag{1.6}$$

$$(\mathbf{u} \otimes \mathbf{v}) \cdot \mathbf{w} = (\mathbf{u} \otimes \mathbf{v}) \mathbf{w} = \mathbf{u} (\mathbf{v} \cdot \mathbf{w}) \tag{1.7}$$

Double contraction:

$$(\mathbf{u} \otimes \mathbf{v}) : (\mathbf{w} \otimes \mathbf{z}) = (\mathbf{u} \cdot \mathbf{w})(\mathbf{v} \cdot \mathbf{z}) = \sum_{i,j} u_i v_j w_i z_j \quad (1.8)$$

$$\mathbf{T} : \mathbf{S} = \sum_{i,j} T_{ij} S_{ij} \quad (1.9)$$

Gradient of a vector field:

$$\nabla \mathbf{u} = \sum_{i,j} u_{i,j} \check{e}^{(i)} \otimes \check{e}^{(j)} \quad (1.10)$$

$$(\underline{\nabla \mathbf{u}})_{ij} = u_{i,j} \quad (1.11)$$

Theorem 1.1 *Volume integral of a gradient.*

$$\int_V \varphi_{,i} dV = \int_{\partial V} \varphi n_i dS \quad (1.12)$$

Theorem 1.2 *Gauss-Green, $\check{\mathbf{n}}$ is the outward normal.*

$$\int_V \nabla \cdot \mathbf{z} dV = \int_{\partial V} \mathbf{z} \cdot \check{\mathbf{n}} dS \quad (1.13)$$

Outer product, cross product:

$$\mathbf{w} \times \mathbf{z} = \varepsilon_{ijk} w_j z_k \check{\mathbf{e}}^{(i)} \quad (1.14)$$

Curl of a vector:

$$\nabla \times \mathbf{z} = \varepsilon_{ijk} z_{k,j} \check{\mathbf{e}}^{(i)} \quad (1.15)$$

Exo. 1.1 Show that the divergence of $\nabla \times \mathbf{z}$ is zero, for any differentiable vector field \mathbf{z} . Show that the curl of $\nabla\varphi$ is zero, for any differentiable scalar function φ .

Exo. 1.2 Let V be a connected volume in $3D$, with boundary ∂V . Assume that the fluid inside V is at constant pressure, exerting a force

$$\mathbf{F} = p \check{\mathbf{n}} \quad (1.16)$$

per unit area on ∂V . Prove that the total force exerted by the inner fluid on the boundary is zero.

Exo. 1.3 Let V be a volume in $3D$, with boundary ∂V . Assume the volume is filled with a fluid of constant density ρ . Prove that the total weight can be obtained from surface integrals:

$$\int_V \rho g \, dV = \frac{\rho g}{3} \int_{\partial V} \mathbf{x} \cdot \check{\mathbf{n}} \, dS = \rho g \int_{\partial V} x_3 n_3 \, dS \quad (1.17)$$

Exo. 1.4 Prove Archimedes' principle. A body immersed in a stagnant homogeneous liquid (which has pressure proportional to its depth, $p = \rho g h$) experiences a net upward force equal to the weight of the displaced liquid.

1.3 Kinematics, material derivative and transport theorem

The trajectory of particles in a continuum can be described by a function $\mathcal{F}(\mathbf{x}, s, t)$ which gives *the position at time t of the particle that occupies position \mathbf{x} at time s* .

- $\mathcal{F}(\mathbf{x}, t, t) = \mathbf{x}$ for all t .
- Fixing s and t , considered just as function of \mathbf{x} , the function $\phi(\mathbf{x}) = \mathcal{F}(\mathbf{x}, s, t)$ is the *deformation* field of the medium between times s and t .
- The velocity field is related to \mathcal{F}

$$\frac{\partial \mathcal{F}}{\partial t}(\mathbf{x}, s, t) = \mathbf{u}(\mathcal{F}(\mathbf{x}, s, t), t) \quad (1.18)$$

Here the pair (\mathbf{x}, s) are a label for the *particle*. Another usual label is \mathbf{X} , defined as the position occupied by the particle in some “reference configuration”, which needs not correspond to an instant of time. This is the so-called Lagrangian frame.

- Trajectories are sometimes written as

$$\mathbf{x}(t) = \phi(\mathbf{X}, t) \quad (1.19)$$

- Pathlines, streamlines and streaklines.

Exo. 1.5 *A continuum is rigidly rotating with angular velocity ω around the axis $\mathbf{a} = \check{\mathbf{e}}^{(1)} + \check{\mathbf{e}}^{(2)}$. Compute its Eulerian velocity field $\mathbf{u}(\mathbf{x}, t)$ and its kinematic history function $\mathcal{F}(\mathbf{x}, s, t)$.*

The *material* or *total* derivative of a quantity ψ at time t for the particle that at that time is located at \mathbf{x} is defined as the “derivative following the particle”, or, more precisely,

$$\frac{D\psi}{Dt} = \lim_{\delta \rightarrow 0} \frac{\psi(\mathcal{F}(\mathbf{x}, t, t + \delta), t + \delta) - \psi(\mathbf{x}, t)}{\delta} \quad (1.20)$$

Exo. 1.6 Prove that

$$\frac{D\psi}{Dt} = \partial_t \psi + \mathbf{u} \cdot \nabla \psi \quad (1.21)$$

The *acceleration* of a fluid is the material derivative of the velocity

$$\mathbf{a} = \frac{D\mathbf{u}}{Dt} = \partial_t \mathbf{u} + (\mathbf{u} \cdot \nabla) \mathbf{u} = \partial_t \mathbf{u} + (\nabla \mathbf{u}) \cdot \mathbf{u} \quad (1.22)$$

Exo. 1.7 Compute the acceleration field of the rigid rotation described in Exo. 1.5.

Let Ω be a region in space, and let $f(\mathbf{x}, t)$ be a scalar field defined in Ω . To fix ideas, let f be a *temperature* field.

Let us select, at time t , a region V of Ω . This defines a *material volume*, consisting of the set of material particles that are inside V at time t .

If one follows the particles that are in V at t , they will occupy another region of space $\mathcal{V}(t')$ at time t' . Obviously $\mathcal{V}(t) = V$.

For any t' , let $I(t')$ be the integral of f , at time t' , over the volume occupied $\mathcal{V}(t')$ by the particles

$$I(t') = \int_{\mathcal{V}(t')} f(\mathbf{x}, t') dV . \quad (1.23)$$

Clearly $I(t')$ is the integral of the temperature over the material volume, a volume that changes position in time but has fixed material identity.

Reynolds transport theorem.

$$\frac{DI}{Dt}(t) = \int_V [\partial_t f + \nabla \cdot (\mathbf{u} f)] dV = \int_V \partial_t f dV + \int_{\partial V} f \mathbf{u} \cdot \mathbf{\check{n}} dS \quad (1.24)$$

Exo. 1.8 Use the previous formula to prove that a flow in which the volume of each material part is preserved must be solenoidal ($\nabla \cdot \mathbf{u} = 0$), also called *incompressible*.

Computational exercise:

- Consider a structured mesh in space-time: $\{x_i\} \times \{y_j\} \times \{t_k\}$. Consider that a velocity vector is known on each node and time of the mesh: $\{\mathbf{u}_{ij}^k\}$.
- A velocity field $\mathbf{u}(\mathbf{x}, t)$ is defined by **trilinear interpolation** of the instantaneous nodal velocity vectors.
- Consider also that a set of points $\{\mathbf{X}_m\}$ is given.

Build an Octave code that calculates the trajectories of particles that, at time t_0 , are in the positions $\{\mathbf{X}_m\}$. Plot and animate in an interesting example.

1.4 Conservation of mass

Let M be the mass contained at time t in volume V ,

$$M = \int_V \rho \, dV . \quad (1.25)$$

Since *the mass is conserved*,

$$\frac{DM}{Dt} = 0 , \quad (1.26)$$

which implies that (**integral form**)

$$\int_V \partial_t \rho \, dV = - \int_{\partial V} \rho \mathbf{u} \cdot \check{\mathbf{n}} \, dS \quad (1.27)$$

and also that (**differential form**)

$$\partial_t \rho + \nabla \cdot (\rho \mathbf{u}) = 0 \quad (1.28)$$

This last equation can be written as

$$\frac{D\rho}{Dt} + \rho \nabla \cdot \mathbf{u} = 0 , \quad (1.29)$$

which shows that an incompressible flow ($\nabla \cdot \mathbf{u} = 0$) in which the density of the material particles does not change with time automatically satisfies mass conservation.

The **mass flux** is given by

$$\mathbf{j} = \rho \mathbf{u} . \quad (1.30)$$

The conservation of mass can be written as a *conservation law*:

$$\partial_t \rho + \nabla \cdot \mathbf{j} = g \quad (1.31)$$

where g represents the sources (in the case of mass equal to zero).

$$\frac{d}{dt} \int_V \rho \, dV = - \int_{\partial V} \underbrace{\mathbf{j} \cdot \hat{\mathbf{n}}}_J \, dS + \int_V g \, dV \quad \text{variation} = \text{inflow} - \text{outflow} + \text{internal sources} \quad (1.32)$$

Exo. 1.9 Let ψ be the mass density, or mass fraction, of some species A dispersed in the medium. The mass of this species in some volume V is

$$M_A = \int_V \rho \psi \, dV . \quad (1.33)$$

Derive conservation laws in differential and integral form for ψ . Also prove that

$$\frac{D\psi}{Dt} = 0 . \quad (1.34)$$

1.5 Conservation of momentum

The total momentum contained by a region V of a continuum is

$$\mathbf{P} = \int_V \rho \mathbf{u} dV . \quad (1.35)$$

The principle of conservation of momentum states that changes in the momentum are equal to the applied (volumetric and surface) forces, i.e.

$$\frac{D\mathbf{P}}{Dt} = \int_V \mathbf{f} dV + \int_S \mathbf{F} dS . \quad (1.36)$$

Using the transport theorem one arrives at the integral form

$$\frac{d}{dt} \int_V \rho \mathbf{u} dV = \int_V \mathbf{f} dV + \int_{\partial V} [\mathbf{F} - \rho (\mathbf{u} \otimes \mathbf{u}) \mathbf{\check{n}}] dS . \quad (1.37)$$

The Cauchy stress tensor

The *action-reaction principle* requires that, if at a point \mathbf{x} of ∂V the region is subject to a surface force density $\mathbf{F}(\mathbf{x})$, the continuum inside reacts with an equal and opposite force.

It can be proved that there exists a symmetric tensor, the *Cauchy stress tensor*, such that for all \mathbf{x} and t

$$\mathbf{F}(\mathbf{x}, t) = \boldsymbol{\sigma}(\mathbf{x}, t) \cdot \check{\mathbf{n}}(\mathbf{x}, t) , \quad (1.38)$$

in the sense that *the surface forces that a medium exerts on another body through a surface with normal \mathbf{n} (pointing outwards) is equal to $-\boldsymbol{\sigma} \cdot \check{\mathbf{n}}$.*

Inserting the stress tensor in (1.37) one arrives at

$$\frac{d}{dt} \int_V \rho \mathbf{u} \, dV = \int_V \mathbf{f} \, dV + \int_{\partial V} (\boldsymbol{\sigma} - \rho \mathbf{u} \otimes \mathbf{u}) \cdot \check{\mathbf{n}} \, dS . \quad (1.39)$$

The momentum flux through a surface is, thus,

$$\boldsymbol{\zeta} = -\boldsymbol{\sigma} + \rho \mathbf{u} \otimes \mathbf{u} \quad (1.40)$$

Exo. 1.10 From (1.39) deduce the following differential forms of momentum conservation:

Conservative form:

$$\partial_t(\rho \mathbf{u}) + \nabla \cdot \boldsymbol{\zeta} = \mathbf{f} \quad \text{or} \quad (1.41)$$

$$\partial_t(\rho \mathbf{u}) + \nabla \cdot (\rho \mathbf{u} \otimes \mathbf{u}) = \nabla \cdot \boldsymbol{\sigma} + \mathbf{f} \quad (1.42)$$

Non-conservative form:

$$\rho \partial_t \mathbf{u} + \rho (\mathbf{u} \cdot \nabla) \mathbf{u} = \nabla \cdot \boldsymbol{\sigma} + \mathbf{f} \quad (1.43)$$

Also, write the equations above in Cartesian components.

1.6 Conservation of energy

Exo. 1.11 Read 1.6 and 1.7 from Wesseling.

The energy of a part of a continuum which occupies volume V is

$$E = \int_V \rho \left(\frac{1}{2} |\mathbf{u}|^2 + e \right) dV \quad (1.44)$$

where e is the *internal energy per unit mass*, which expresses the capability of a medium storing energy and is a function of its *local state*. The principle of conservation of energy reads

$$\frac{DE}{Dt} = \mathcal{Q} + \mathcal{W} , \quad (1.45)$$

where the right-hand side is the sum of the heat and work received from the surroundings. Defining \mathbf{q} as the heat flux and Q as the heat source per unit volume one gets

$$\frac{DE}{Dt} = \int_V (\mathbf{f} \cdot \mathbf{u} + Q) dV + \int_{\partial V} (\mathbf{u} \cdot \boldsymbol{\sigma} - \mathbf{q}) \cdot \mathbf{\check{n}} dS \quad (1.46)$$

Exo. 1.12 From the equation above, prove the following differential form

$$\rho \frac{De}{Dt} = -\nabla \cdot \mathbf{q} + \boldsymbol{\sigma} : \nabla \mathbf{u} + Q \quad (1.47)$$

1.7 Constitutive laws

If one counts the equations up to now we have

- Conservation of mass (1 equation).
- Conservation of momentum (3 equations).
- Conservation of energy (1 equation).

Total: **5 equations**.

Counting the unknowns: ρ (1), \mathbf{u} (3), $\boldsymbol{\sigma}$ (6), e (1), \mathbf{q} (3). Total: **14 unknowns**.

The 9 equations that are lacking come from the so-called *constitutive laws*, that describe the material behavior (notice that the equations up to now hold for *any* continuum).

Essentially we need laws for e , $\boldsymbol{\sigma}$ and \mathbf{q} . For the latter Fourier's law is almost universally adopted,

$$\mathbf{q} = -\boldsymbol{\kappa} \nabla T, \quad (1.48)$$

where T is the temperature and $\boldsymbol{\kappa}$ the thermal conductivity (in general a tensor).

1.8 Newtonian and quasi-newtonian behavior

- The stress of a fluid at a point \mathbf{x} and instant t can in principle depend on the whole deformation history of the vicinity of \mathbf{x} .
- However, not all constitutive laws correspond to fluids. The definition of fluid requires that “if the vicinity of the point has not deformed at all, then the stress tensor must be spherical”. Spherical, in this context, means that $\boldsymbol{\sigma}$ is a multiple of the identity.
- A most important class of fluid constitutive laws corresponds to the so-called *quasi-Newtonian fluids*:

$$\boldsymbol{\sigma} = (-p + \lambda \nabla \cdot \mathbf{u}) \mathbf{1} + \mu (\nabla \mathbf{u} + \nabla \mathbf{u}^T) \quad (1.49)$$

in which λ and μ can depend on the *instantaneous deformation rate tensor*

$$\boldsymbol{\varepsilon}(\mathbf{u}) = D \mathbf{u} = \frac{1}{2} (\nabla \mathbf{u} + \nabla \mathbf{u}^T) . \quad (1.50)$$

- Since λ and μ are scalars, the model is *objective* only if they depend on $\boldsymbol{\varepsilon}(\mathbf{u})$ through its *invariants*:

$$I = \text{trace } \boldsymbol{\varepsilon}(\mathbf{u}) = \mathbf{1} : \boldsymbol{\varepsilon}(\mathbf{u}) = \nabla \cdot \mathbf{u} \quad (1.51)$$

$$II = \frac{1}{2} [(\text{trace } \boldsymbol{\varepsilon}(\mathbf{u}))^2 - \boldsymbol{\varepsilon}(\mathbf{u}) : \boldsymbol{\varepsilon}(\mathbf{u})] \quad (1.52)$$

$$III = \det \boldsymbol{\varepsilon}(\mathbf{u}) \quad (1.53)$$

Notice that, in particular, the *deformation rate*

$$\|\boldsymbol{\varepsilon}(\mathbf{u})\| = \sqrt{\boldsymbol{\varepsilon}(\mathbf{u}) : \boldsymbol{\varepsilon}(\mathbf{u})} \quad (1.54)$$

- If λ and μ are constants, eventually dependent on the temperature, the fluid is called *Newtonian*.

- Shear thinning (resp. shear thickening) describe fluids in which μ is a decreasing (resp. increasing) function of $\|\boldsymbol{\varepsilon}(\mathbf{u})\|$.

Exo. 1.13 Knowing that the velocity field of a rigid body motion is given by

$$\mathbf{u}(\mathbf{x}, t) = \mathbf{z}(t) + \mathbf{r}(t) \times \mathbf{x} , \quad (1.55)$$

1. Prove that $\boldsymbol{\varepsilon}(\mathbf{u})$ is zero.
2. Compute the **vorticity** $\boldsymbol{\omega} = \nabla \times \mathbf{u}$ and find its relation to \mathbf{r} and to the **antisymmetric part of the velocity gradient**, $\nabla^A \mathbf{u} = \frac{1}{2} (\nabla \mathbf{u} - \nabla \mathbf{u}^T)$.

Exo. 1.14 For an incompressible fluid, the term $\Phi = \boldsymbol{\sigma} : \nabla \mathbf{u}$ in the differential equation **dissipation of energy**, i.e., the power transformed into heat. Write down Φ in Cartesian coordinates.

1.9 Boundary conditions

Exo. 1.15 Read 1.6 from Kirby.

Exo. 1.16 Read, fill in the details and reproduce (part of) the results of the articles by N. Morhell and H. Pastoriza (Microfluidics and Nanofluidics, 2013, Sensors and Actuators B, 2016).

2 Brief overview of numerical methods for CFD

For this chapter we follow basically the two references:

- Finite Volume Methods. R. Eymard, T. Gallouët and R. Herbin. 2003. Pages 4-26, and also some small parts of Chapter 3.
- Principles of Computational Fluid Dynamics. P. Wesseling. 2001. Chapter 3.

2.1 Differential, integral and variational formulations

Consider the general second-order differential equation

$$L\varphi = -(\mathbf{a}_{ij}\varphi_{,j})_{,i} + (\mathbf{b}_i\varphi)_{,i} + c\varphi = q . \quad (2.1)$$

This equation is said to be *uniformly elliptic* if there exists $C > 0$ such that

$$\mathbf{v} \cdot (\mathbf{a}(\mathbf{x}) \cdot \mathbf{v}) = a_{ij}(\mathbf{x}) v_i v_j \geq C \|\mathbf{v}\|^2 \quad \forall \mathbf{x} \quad \forall \mathbf{v} . \quad (2.2)$$

This condition, together with suitable boundary conditions, guarantees the existence of a unique φ in the space $H^1(\Omega)$. This solution is continuous (a.e.) across any surface.

Equation (2.1) can be seen as a steady conservation law in differential formulation,

$$\nabla \cdot \mathbf{j} = g , \quad (2.3)$$

by taking

$$\mathbf{j} = \mathbf{J}(\varphi, \nabla\varphi) = -\mathbf{a} \nabla\varphi + \mathbf{b} \varphi \quad (2.4)$$

and

$$g = q - c\varphi . \quad (2.5)$$

There thus exists a unique $\varphi \in H^1(\Omega)$ that satisfies the boundary conditions and also (2.3) for all \mathbf{x} in the domain Ω of the problem. This is the **differential formulation**, which is the start point of **finite difference** approximation methods.

The differential equation must be understood in a *weak sense*, i.e.,

$$-\int_{\Omega} \mathbf{j} \cdot \nabla \psi \, dV + \int_{\partial\Omega} \psi \mathbf{j} \cdot \check{\mathbf{n}} \, dS = \int_{\Omega} g \psi \, dV \quad (2.6)$$

for all $\psi \in H^1(\Omega)$. Notice that this formula has no derivative of \mathbf{j} and thus makes sense in cases in which the strong form (2.3) does not.

Considering homogeneous Dirichlet boundary conditions, the **variational formulation** of the problem reads: “Find $\varphi \in H_0^1(\Omega)$ such that

$$-\int_{\Omega} \mathbf{J}(\varphi, \nabla \varphi) \cdot \nabla \psi \, dV = \int_{\Omega} g(\varphi) \psi \, dV \quad (2.7)$$

for all $\psi \in H_0^1(\Omega)$.”

This formulation is adopted in **primal finite element methods**, in which φ_h belongs to some subspace V_h and satisfies (2.7) only for functions ψ belonging to V_h .

Let Γ be a surface that divides Ω into two parts, Ω_1 and Ω_2 . Integrating by parts (2.7) in each Ω_i one obtains

$$\int_{\Omega_1} [\nabla \cdot \mathbf{J}(\varphi, \nabla \varphi) - g(\varphi)] \psi \, dV + \int_{\Omega_2} [\nabla \cdot \mathbf{J}(\varphi, \nabla \varphi) - g(\varphi)] \psi \, dV - \int_{\Gamma} \llbracket \mathbf{J}(\varphi, \nabla \varphi) \cdot \mathbf{\check{n}} \rrbracket \psi \, dS = 0 \quad \forall \psi \in H_0^1(\Omega). \quad (2.8)$$

This implies that

- The solution of (2.7) satisfied the differential equation a.e. in Ω_1 and Ω_2 .
- The normal flux $\mathbf{J} \cdot \mathbf{\check{n}}$ is continuous across Γ .

Exo. 2.1 *Give arguments to support (or prove) both previous statements.*

Let K be an open polyhedral subset of Ω , with facets $e \in \mathcal{E}$. Integrating (2.3) over K and using Gauss-Green formula one gets

$$\sum_{e \in \partial K} \int_e \mathbf{J}(\varphi, \nabla \varphi) \cdot \mathbf{n} \, dS = \int_K g(\varphi) \, dK . \quad (2.9)$$

Notice that $\mathbf{J} \cdot \mathbf{n}$ is well defined on e . The **integral formulation** of the problem corresponds to “find the unique $\varphi \in H^1(\Omega)$ such that (2.9) holds for all polyhedra K contained in Ω ”.

- The integral formulation is the basis of **finite volume methods**. The discretization methodology consists of selecting a finite number of polyhedra as the finite volume mesh \mathcal{T}_h , and obtaining a finite number of equations by only requiring that (2.9) holds for those polyhedra. This leads to

$$\sum_{e \in \partial K} \bar{F}_{K,e} = \int_K g \, dV \quad \forall K \in \mathcal{T}_h . \quad (2.10)$$

- The next step is the selection of degrees of freedom for the discrete solution. The most usual choice is to have one unknown φ_K per finite volume K , i.e., N_V unknowns for N_V equations. In addition, a node \mathbf{x}_K is defined for each K .
- Letting $\underline{\varphi} \in \mathbb{R}^{N_V}$ be the column array of unknowns, a **numerical flux function** $F_{K,e}(\underline{\varphi})$ is introduced satisfying a **consistency condition**

$$F_{K,e}(\underline{\varphi}^*) \simeq \bar{F}_{K,e}(\varphi, \nabla \varphi) \quad (2.11)$$

where $\underline{\varphi}^* = (\varphi(\mathbf{x}_1), \varphi(\mathbf{x}_2), \dots)^T$ is the array of nodal values of any **exact** solution φ of the problem.

- The discrete system of equations reads

$$\sum_{e \in \partial K} F_{K,e}(\underline{\varphi}) = \int_K g(\underline{\varphi}) \, dV \quad \forall K \in \mathcal{T}_h . \quad (2.12)$$

- The finite volume method extends naturally to transient problems. If the equation considered is

$$\partial_t \phi + L \phi = q , \quad (2.13)$$

then upon FV discretization in space one ends up with

$$V_K \frac{d\varphi_K}{dt} + \sum_{e \in \partial K} F_{K,e}(\underline{\varphi}) = \int_K g(\underline{\varphi}) dV \quad \forall K \in \mathcal{T}_h . \quad (2.14)$$

Above, V_K is the volume of cell K . The numerical problem thus reduces to a system of ODE that is then discretized in time with a variety of methods.

- For the method to be strictly conservative, it must happen that if a given facet e separates cell K from cell L then

$$F_{K,e}(\underline{\varphi}) = -F_{L,e}(\underline{\varphi}) . \quad (2.15)$$

- An interesting alternative to our choice of degrees of freedom is to add an additional unknown per facet. Let \mathcal{E} be the “skeleton” of the mesh, consisting of all facets e , and let $\hat{\varphi}_j$, with $j = 1, \dots, N_E$ be the facet unknowns. One now has N_V equations and $N_V + N_E$ unknowns. The required additional equations are (2.15), closing the system.
- Other possibilities exist, such as overlapping finite volumes, but we will not discuss them here.

2.2 A one-dimensional example

Let us take

$$L\varphi = -(\mathbf{a} \phi_{,1})_{,1} = q \quad (2.16)$$

in the domain $(0, \ell)$, which has nodes $0 = x_0, x_1, \dots, x_n = \ell$. Let $h_i = x_i - x_{i-1}$. Also, let $x_{i+\frac{1}{2}} = \frac{1}{2}(x_i + x_{i+1})$ and $h_{i+\frac{1}{2}} = \frac{1}{2}(h_i + h_{i+1})$.

Finite differences

$$\begin{aligned} (\mathbf{a} \varphi')'(x_j) &\simeq \frac{\mathbf{a}(x_{j+\frac{1}{2}}) \varphi'(x_{j+\frac{1}{2}}) - \mathbf{a}(x_{j-\frac{1}{2}}) \varphi'(x_{j-\frac{1}{2}})}{h_{j+\frac{1}{2}}} \\ &\simeq \frac{\frac{\mathbf{a}_j + \mathbf{a}_{j+1}}{2} \frac{\varphi(x_{j+1}) - \varphi(x_j)}{h_{j+1}} - \frac{\mathbf{a}_{j-1} + \mathbf{a}_j}{2} \frac{\varphi(x_j) - \varphi(x_{j-1})}{h_j}}{h_{j+\frac{1}{2}}}. \end{aligned} \quad (2.17)$$

For equispaced nodes this leads to the discrete scheme (3.9) of Wesseling.

Exo. 2.2 *Build a small code for this problem and solve the interface problem of page 84 of Wesseling. Compare to the results shown in the book.*

Finite volumes

Notice that $J(\varphi, \varphi') = -a \varphi'$. Letting the finite volumes be given by $V_j = (x_{j-\frac{1}{2}}, x_{j+\frac{1}{2}})$ a reasonable numerical flux (for continuous a) is

$$F_{j+\frac{1}{2}} = -\frac{a_j + a_{j+1}}{2} \frac{\varphi_{j+1} - \varphi_j}{h_{j+1}}. \quad (2.18)$$

Exo. 2.3 *Build the corresponding finite volume method in terms of nodal quantities. Compare to the finite-difference scheme.*

Improved finite volumes

Let us introduce as additional degrees of freedom the values $\varphi_{j+\frac{1}{2}}$ and

$$F_{j,j+\frac{1}{2}} = -a_j \frac{\varphi_{j+\frac{1}{2}} - \varphi_j}{h_{j+1}/2}. \quad (2.19)$$

Similarly, we have

$$F_{j+1,j+\frac{1}{2}} = a_{j+1} \frac{\varphi_{j+1} - \varphi_{j+\frac{1}{2}}}{h_{j+1}/2}. \quad (2.20)$$

Conservation condition (2.15) then allows to eliminate the unknown $\varphi_{j+\frac{1}{2}}$,

$$F_{j+\frac{1}{2}} = F_{j,j+\frac{1}{2}} = -F_{j+1,j+\frac{1}{2}} \quad \Rightarrow \quad \varphi_{j+\frac{1}{2}} = \frac{a_j \varphi_j + a_{j+1} \varphi_{j+1}}{a_j + a_{j+1}}. \quad (2.21)$$

Exo. 2.4 *Build the finite volume scheme corresponding to the flux above. Compare to (3.17) de Wesseling. Modify the code of exercise 2.2 to implement it. Test it. Compute the convergence order in a smooth problem with analytical solution.*

Exo. 2.5 Study and discuss cell-centered finite volumes for the 1D problem, in which the nodes are $x_{j+\frac{1}{2}}$ instead of x_j and the finite volumes are of the form (x_j, x_{j+1}) . Modify the code to deal with cell-centered discretization and compare to previous results.

Exo. 2.6 Analyze the consistency (truncation error) of the fluxes and of the overall stencil of the vertex-centered scheme of Exo. 2.3. Consider $a \equiv 1$, $f = 1$ and h_i equal to h if i is even and equal to $h/2$ when i is odd. Discuss the result together with a numerical experiment.

Exo. 2.7 Study Chapter 2 (and part of Chapter 3) of Eymard et al's "Finite Volume Methods":

1. What is the definition of an admissible one-dimensional mesh? Are cell-centered and vertex-centered meshes admissible?
2. Do the calculations showing that a cell-centered scheme is not consistent in the usual finite-difference sense (Example 2.1 and Remark 2.3).
3. Follow step by step the proof of Theorem 2.1.
4. Do the calculations that lead to equation 2.26 and to the harmonic mean formula of Example 2.2.
5. Follow step by step the proof of Theorem 2.3.
6. What is an admissible mesh in 2D? Give examples of admissible meshes and of inadmissible meshes. What is a Voronoïmesh? Are Voronoïmeshes always admissible? Why?
7. Explain Equation 3.86: What is the equation corresponding to a cell that has its boundary at the boundary of Ω ?
8. Explain in simple words Definition 3.7 of Neumann restricted admissible meshes.

Exo. 2.8 (Miniproject) Consider a microchannel with (electrically) non-conducting walls and a conducting fluid. The geometry is given by a mask on a rectangular mesh, so that the mask takes different values depending on the cell being fluid, wall, inlet, outlet. The electric potential satisfies

$$\Delta\Phi = 0 \tag{2.22}$$

in the fluid, with $\partial\Phi/\partial n = 0$ at the walls, and Φ given at inlets and outlets. Code a finite volume solver for the electric potential and compute from it the electric field $\mathbf{E} = -\nabla\Phi$.

In electro-osmotic flows with homogeneous material properties the fluid velocity satisfies

$$\mathbf{u} = -\kappa \mathbf{E} \tag{2.23}$$

where κ is a material constant. With the computed electric field simulate the transport of inert particles for a non-straight microchannel.

Also, take a look at the video

<https://br.comsol.com/video/simulating-electrokinetic-phenomena-microfluidics#>

3 Numerical approximation of fully developed flow

3.1 The physical setting

- Incompressible flow along a long cylinder of cross section $\Omega \subset \mathbb{R}^2$. The flow domain is $\mathcal{B} = \Omega \times (0, L)$.
- The flow is driven by a pressure gradient

$$\mathcal{G} = \frac{p(L) - p(0)}{L} \quad (3.1)$$

notice that when $\mathcal{G} > 0$ we expect $w = u_3 < 0$ and viceversa.

- If L is sufficiently large, the entry and exit effects can be neglected and all cross sections are essentially identical, except for the pressure.
- Decomposing the stress tensor in pressure and non-pressure components, we assume

$$\boldsymbol{\sigma}(x_1, x_2, x_3, t) = -p(x_3, t)\mathbb{I} + \boldsymbol{\sigma}^*(x_1, x_2, t) . \quad (3.2)$$

- Let ω be an arbitrary region in Ω and let V be the corresponding cylinder, i.e.,

$$V = \omega \times (0, L) . \quad (3.3)$$

We denote also $\omega_z = \omega \times \{z\}$ (the cross section at $x_3 = z$) and $\mathcal{S} = \partial\omega \times (0, L)$ (the lateral surface) so that

$$\partial V = \omega_0 \cup \mathcal{S} \cup \omega_L . \quad (3.4)$$

3.2 Conservation principles

- Mass: Because of incompressibility, and assuming ρ is a constant, this principle reads

$$0 = \int_{\partial V} \mathbf{u} \cdot \check{\mathbf{n}} \, dS = - \int_{\omega_0} w \, dS + \int_{\omega_L} w \, dS + \int_S \mathbf{u} \cdot \check{\mathbf{n}} \, dS . \quad (3.5)$$

This condition is automatically satisfied in *parallel flows* which we consider hereafter, i.e., flows in which the velocity is of the form

$$\mathbf{u}(x_1, x_2, x_3, t) = (0, 0, w(x_1, x_2, t)) . \quad (3.6)$$

- Momentum: In parallel flows,

$$L \frac{d}{dt} \int_{\omega} \rho w \, d\omega = -\mathcal{G} L |\omega| + L \int_{\partial\omega} \boldsymbol{\tau} \cdot \check{\boldsymbol{\nu}} \, d\partial\omega \quad (3.7)$$

where

$$\underline{\boldsymbol{\tau}} = (\sigma_{13}, \sigma_{23})^T \quad \text{and} \quad \check{\boldsymbol{\nu}} = (\mathbf{n}_1, \mathbf{n}_2)^T . \quad (3.8)$$

In incompressible isothermal flows the mass and momentum conservation principles form a closed system.

In this case one equation, which is (3.7), in one unknown w .

3.3 Boundary conditions

Exo. 3.1 Read Section 1.6 and Chapter 2 of Kirby.

1. Prove Eq. 1.59.
2. What is the Navier slip boundary condition? Give a physical argument that determines the sign of b .
3. Solve the unidirectional flow between two parallel plates located at $x_3 = 0$ and $x_3 = h$, subject to a pressure gradient $\nabla p = (\partial_1 p, \partial_2 p)$ and with the upper plate moving at a velocity $\mathbf{u} = (U, 0)$ with respect to the lower one. Compute the flux $\mathbf{j} = \int_0^h \mathbf{u} \, dx_3$ as a function of ∇p , U and h (and of the fluid viscosity μ).
4. Justify the claim $\nabla \cdot \mathbf{j} = \partial_1 j_1 + \partial_2 j_2 = 0$, and use this claim to arrive at the **lubrication equation** (also known as Reynolds equation).

- The **no-slip boundary condition** holds when a fluid is in contact with a solid surface, in this case it translates to

$$w(x_1, x_2, t) = 0 \quad \forall (x_1, x_2) \in \partial\Omega . \quad (3.9)$$

- Under certain conditions, the fluid has been observed to **slip** at the solid boundary (e.g., in very rarefied flows). In the parallel flows we are considering, the adopted (Navier) condition amounts to

$$\boldsymbol{\tau} \cdot \check{\boldsymbol{\nu}} = -b(w - w_{\text{wall}}) . \quad (3.10)$$

- If an electric field is applied along the channel, then a non-zero velocity “appears” at the wall (read Chapter 6 of Kirby to understand why). This is an apparent wall velocity, which in fact only takes place at a finite distance $\sim 5 \lambda_D$ from it, where λ_D is the **Debye-Hückel length**. Its value is given by the **Helmholtz-Smoluchowski** equation

$$w = m_{eo} E_{\text{wall}} \quad (3.11)$$

where m_{eo} is the **electroosmotic mobility** and is a property of the fluid and the surface material (of the order of $10^{-8} \text{m}^2 / (\text{V s})$).

3.4 Viscous parallel flow

If the fluid is Newtonian-like (Boussinesq),

$$\boldsymbol{\sigma}^* = \mu \begin{pmatrix} 0 & 0 & w_{,1} \\ 0 & 0 & w_{,2} \\ w_{,1} & w_{,2} & 0 \end{pmatrix} \Rightarrow \boldsymbol{\tau} = \mu \nabla w . \quad (3.12)$$

We can, applying Gauss-Green theorem, rewrite (3.7) as

$$\int_{\omega} [\rho \partial_t w + \mathcal{G} - \nabla \cdot (\mu \nabla w)] d\omega = 0 \quad (3.13)$$

and arrive at the differential form (with no-slip conditions for example)

$$\begin{cases} \rho \partial_t w + \mathcal{G}(t) - \nabla \cdot (\mu \nabla w) = 0 & \text{in } \Omega , \\ w = 0 & \text{on } \partial\Omega . \end{cases} \quad (3.14)$$

Writing it as a conservation law

$$\partial_t(\rho w) + \nabla \cdot \mathbf{j} = g , \quad \mathbf{j} = -\mu \nabla w , \quad g = -\mathcal{G} . \quad (3.15)$$

3.5 Discretization in Cartesian grids

3.5.1 Finite differences

Consider a rectangular pipe $\Omega = (0, L_1) \times (0, L_2)$ with a uniform vertex-centered Cartesian grid with nodes at positions

$$\mathbf{X}_{j_1 j_2} = ((j_1 - 1)h_1, (j_2 - 1)h_2), \quad j_\alpha = 1, \dots, n_\alpha + 1, \quad \alpha \in \{1, 2\}, \quad (3.16)$$

where n_α is the number of subdivisions in the α direction and $n_\alpha h_\alpha = L_\alpha$.

Considering as unknowns the values at the nodes w_{j_1, j_2} , we have $w_{j_1, j_2} = 0$ if (j_1, j_2) is at the boundary. For an internal node, on the other hand, a FD space discretization of (3.14) with constant density and viscosity leads to

$$\rho \frac{d}{dt} w_{j_1, j_2} + \mathcal{G} - \mu \frac{w_{j_1+1, j_2} - 2w_{j_1, j_2} + w_{j_1-1, j_2}}{h_1^2} - \mu \frac{w_{j_1, j_2+1} - 2w_{j_1, j_2} + w_{j_1, j_2-1}}{h_2^2} = 0. \quad (3.17)$$

Our first issue is the implementation of this method.

Node-to-unknown mapping:

There are $(n_1 + 1) \times (n_2 + 1)$ unknowns, they can be numbered by row or by column (or else) to get the mapping. Denoting $N_1 = n_1 + 1$, $N_2 = n_2 + 1$,

```
function ng = ij2n (i,j)
    global N1 N2
    ng = i + (j-1)*N1;
endfunction
```

Exo. 3.2 *Build a function n2ij(n) that is the inverse of the previous one.*

Viscous matrix:

$$pP=ij2n(i,j); pN=ij2n(i,j+1); pE=ij2n(i+1,j); pS=ij2n(i,j-1); pW=ij2n(i-1,j);$$

The following matrix row provides the viscous contribution $(L_\mu w)_P \simeq -\mu \nabla^2 w(P)$ to equation P (interior node):

$$\begin{aligned} \text{aux1} &= \mu/dx^2; \text{aux2} = \mu/dy^2; \\ A(pP,pP) &= 2*(\text{aux1}+\text{aux2}); \\ A(pP,pN) &= -\text{aux2}; A(pP,pS) = -\text{aux2}; \\ A(pP,pE) &= -\text{aux1}; A(pP,pW) = -\text{aux1}; \end{aligned}$$

so that

$$-\mu \frac{w_{j_1+1,j_2} - 2w_{j_1,j_2} + w_{j_1-1,j_2}}{h_1^2} - \mu \frac{w_{j_1,j_2+1} - 2w_{j_1,j_2} + w_{j_1,j_2-1}}{h_2^2} = (\underline{\underline{A}} \underline{W})_P \quad . \quad (3.18)$$

Considering just the interior nodes, we get the system

$$\rho \frac{d}{dt} \underline{W} + \underline{\underline{A}} \underline{W} = \underline{b}(t) \quad (3.19)$$

where $b_P(t) = -G(t)$. Discretizing now in time by the θ -method,

$$\left(\frac{\rho}{\Delta t} \underline{\underline{I}} + \theta \underline{\underline{A}} \right) \underline{W}^{n+1} = \left(\frac{\rho}{\Delta t} \underline{\underline{I}} - (1-\theta) \underline{\underline{A}} \right) \underline{W}^n + \underline{b}^{n+\theta} \quad (3.20)$$

or

$$\underline{\underline{M}} \underline{W}^{n+1} = \underline{\underline{R}} \underline{W}^n + \underline{b}^{n+\theta} \quad (3.21)$$

```

#-- Assembly: loop over nodes
for i=1:N1
    for j=1:N2
        if (i==1 || i==N1 || j==1 || j==N2)
            continue;
        else
# viscous matrix
            pP=ij2n(i,j); pN=ij2n(i,j+1); pE=ij2n(i+1,j); pS=ij2n(i,j-1); pW=ij2n(i-1,j);
            aux1 = mu/dx^2; aux2 = mu/dy^2;
            Af(pP,pP) = 2*(aux1+aux2);
            Af(pP,pN)=-aux2; Af(pP,pS)=-aux2; Af(pP,pE)=-aux1; Af(pP,pW)=-aux1;
# mass matrix
            Am(pP,pP)=rho/dt; bm(pP)=dx*dy;
        endif
    endfor
endfor
#-- Timestepping Matrices: M, R
M = Am + theta*Af;
R = Am - (1-theta)*Af;
#-- Correct M for no-slip boundary conditions
for i=1:N1
    for j=1:N2
        if (i==1 || i==N1 || j==1 || j==N2)
            pP=ij2n(i,j); M(pP,pP)=1;
        endif
    endfor
endfor

```

Exo. 3.3 (Miniproject: electroosmotic pump) Consider a pipe of rectangular cross section $(0, W) \times (0, H)$ and length L , such that the horizontal walls are made of glass and the vertical ones of PDMS. Considering water as the fluid, the corresponding electroosmotic mobilities are $m_{eo} = 3 \times 10^{-8}$ and $1.5 \times 10^{-8} \text{ m}^2/(\text{V}\cdot\text{s})$. The water has $\rho = 1000 \text{ kg/m}^3$ and $\mu = 10^{-3} \text{ Pa}\cdot\text{s}$.

Take $W = 20$ microns, $H = 10$ microns and $L = 3 \text{ mm}$.

Adapt the code `pipe_fd_t.m` and answer the following questions:

1. Considering that both ends of the pipe are at atmospheric pressure, what is the steady flow rate and average velocity for a voltage difference of 1 Volt between the ends of the pipe? What is the shape of the steady velocity profile? How long does it take to reach the steady flow rate?
2. If one end of the pipe is closed, what will be the pressure difference between its ends?

3.6 Vertex-centered finite volumes

- The node-to-unknown mapping remains the same. To allow for variable spacing we assume that arrays $X(1 : N_1)$ and $Y(1 : N_2)$ are given, containing the nodal coordinates.
- From (3.7), the equation for the (interior) finite volume P is

$$F_{PN} + F_{PE} + F_{PS} + F_{PW} = \int_{\omega_P} (-\mathcal{G} - \rho \partial_t w) d\omega \simeq m(\omega_P) \left(-\mathcal{G} - \rho \frac{dW_P}{dt} \right) \quad (3.22)$$

where we have treated $\partial_t w$ as a source and the left-hand side approximates $\int_{\partial\omega_P} \mathbf{j} \cdot \check{\nu} ds$ (remember that $\mathbf{j} = -\mu \nabla w$).

- Now we have to define the discrete fluxes

$$\int_{e_N} \mathbf{j} \cdot \check{\nu} dx_1 = \int_{e_N} j_2 dx_1 = \int_{e_N} (-\mu w_{,2}) dx_1 \simeq -\mu \frac{W_P - W_N}{y_P - y_N} \frac{x_E - x_W}{2} \doteq F_{PN} \quad (3.23)$$

and analogously

$$F_{PE} \doteq -\mu \frac{W_P - W_E}{x_P - x_E} \frac{y_N - y_S}{2} \quad (3.24)$$

$$F_{PS} \doteq \mu \frac{W_P - W_S}{y_P - y_S} \frac{x_E - x_W}{2} \quad (3.25)$$

$$F_{PW} \doteq \mu \frac{W_P - W_W}{x_P - x_W} \frac{y_N - y_S}{2} \quad (3.26)$$

- If we consider the mesh uniform and divide everything by $h_1 h_2$, we arrive at the discrete equation

$$\rho \frac{dW_P}{dt} + \mu \frac{W_P - W_N}{h_2^2} + \mu \frac{W_P - W_S}{h_2^2} + \mu \frac{W_P - W_E}{h_1^2} + \mu \frac{W_P - W_W}{h_1^2} = -\mathcal{G} , \quad (3.27)$$

which shows that balancing fluxes over control volumes indeed leads to a discretization of the Laplacian (equivalent to finite differences, in simple cases).

- In the FV case the inertia matrix is diagonal but not proportional to the identity:

$$B_{PP} = m(\omega_P) \rho . \quad (3.28)$$

- Similarly, the right-hand side is now

$$b_P = - m(\omega_P) \mathcal{G} . \quad (3.29)$$

- The viscous matrix can be built by summing up the contributions of each face:

```
# viscous matrix
  pP=ij2n(i,j); pN=ij2n(i,j+1); pE=ij2n(i+1,j); pS=ij2n(i,j-1); pW=ij2n(i-1,j);
  xP=X(i); xN=xP; xS=xP; xE=X(i+1); xW=X(i-1);
  yP=Y(j); yE=yP; yW=yP; yN=Y(j+1); yS=Y(j-1);
# north face
  aux=mu*(xE-xW)/(yN-yP)/2;
  Af(pP,pP) = Af(pP,pP) + aux;
  Af(pP,pN) = Af(pP,pN) - aux;
# east face
  aux=mu*(yN-yS)/(xE-xP)/2;
  Af(pP,pP) = Af(pP,pP) + aux;
  Af(pP,pE) = Af(pP,pE) - aux;
etcetera
```

- Variable viscosity: Let us assume that the viscosity is not uniform, but a given function $\mu(x, y)$. The modification in the previous code is straightforward:

```

# viscous matrix
  pP=ij2n(i,j); pN=ij2n(i,j+1); pE=ij2n(i+1,j); pS=ij2n(i,j-1); pW=ij2n(i-1,j);
  xP=X(i); xN=xP; xS=xP; xE=X(i+1); xW=X(i-1);
  yP=Y(j); yE=yP; yW=yP; yN=Y(j+1); yS=Y(j-1);
# north face
  xF=xP; yF=(yP+yN)/2;
  aux=mu(xF,yF)*(xE-xW)/(yN-yP)/2;
  Af(pP,pP) = Af(pP,pP) + aux;
  Af(pP,pN) = Af(pP,pN) - aux;
# east face
  xF=(xP+xE)/2; yF=yP;
  aux=mu(xF,yF)*(yN-yS)/(xE-xP)/2;
  Af(pP,pP) = Af(pP,pP) + aux;
  Af(pP,pE) = Af(pP,pE) - aux;
etcetera

```

With the FV formulation we arrive at the system

$$\underline{\underline{B}} \frac{d}{dt} \underline{W} + \underline{\underline{A}} \underline{W} = \underline{b}(t) . \quad (3.30)$$

Discretizing now in time by the θ -method,

$$\left(\frac{1}{\Delta t} \underline{\underline{B}} + \theta \underline{\underline{A}} \right) \underline{W}^{n+1} = \left(\frac{1}{\Delta t} \underline{\underline{B}} - (1 - \theta) \underline{\underline{A}} \right) \underline{W}^n + \underline{b}^{n+\theta} \quad (3.31)$$

or

$$\underline{\underline{M}} \underline{W}^{n+1} = \underline{\underline{R}} \underline{W}^n + \underline{b}^{n+\theta} . \quad (3.32)$$

Quite similar to the FD, uniform spacing case, but now with more general properties and mesh.

- Quasi-newtonian fluid: Viscosity may depend on the shear rate, for incompressible flows given by

$$\dot{\gamma} \doteq \sqrt{D \mathbf{u} : D \mathbf{u}} \quad (3.33)$$

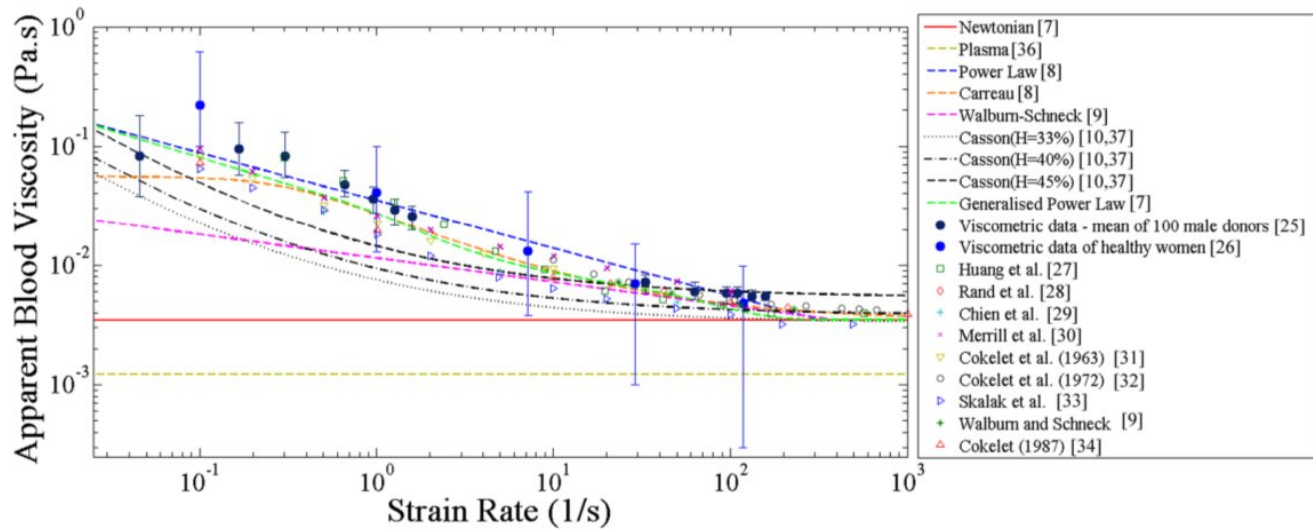


Fig 2. Experimental measurements of blood viscosity and non-Newtonian blood rheological models.

doi:10.1371/journal.pone.0128178.g002

Different models exist for blood

Table 1. Blood rheological model equations.

Blood Model	Effective Viscosity (Pa·s)
Newtonian [7]	$\mu = 0.00345 \text{ Pa}\cdot\text{s}$
Plasma [36]	$\mu = 0.00122 \text{ Pa}\cdot\text{s}$
Power Law (Modified) [8]	$\mu = \begin{cases} m(\dot{\gamma})^{n_p-1}, & \dot{\gamma} < 427 \\ 0.00345 \text{ Pa}\cdot\text{s}, & \dot{\gamma} \geq 427 \end{cases}, m = 0.035, n_p = 0.6$
Walburn-Schneck (Modified) [9]	$\mu = \begin{cases} C_1 e^{(C_2 H)} e^{(C_4 \left(\frac{TPMA}{H^2}\right))} (\dot{\gamma})^{-C_3 H}, & \dot{\gamma} < 414, C_1 = 0.00797, C_2 = 0.0608, C_3 = 0.00499, C_4 = 14.585, H = 40, TPMA = 25.9 \\ 0.00345 \text{ Pa}\cdot\text{s}, & \dot{\gamma} \geq 414 \end{cases}$
Casson [10,37]	$\mu = 0.1 \left(\left[\sqrt{\eta} + \sqrt{\tau_y \left(\frac{1-e^{-m \dot{\gamma} }}{ \dot{\gamma} } \right)} \right]^2 \right), \tau_y = (0.625H)3, \eta = \eta_0(1-H)^{-2.5}, \eta_0 = 0.012, H = 40\% \text{ (female normal), } 33\% \text{ (post-angioplasty) or } 45\% \text{ (male normal)}$
Carreau [8]	$\mu = \mu_{\infty C} + (\mu_0 - \mu_{\infty C}) [1 + (\lambda \dot{\gamma})^2]^{-\frac{n_C-1}{2}}, \lambda = 3.313, n_C = 0.3568, \mu_0 = 0.056, \text{ and } \mu_{\infty C} = 0.00345$
Generalised Power Law [7]	$\mu = \lambda \dot{\gamma} ^{n-1}, \lambda = \mu_{\infty G} + \Delta \mu \exp \left[- \left(1 + \frac{ \dot{\gamma} }{a} \right) \exp \left(- \frac{b}{ \dot{\gamma} } \right) \right], n = n_{\infty} - \Delta n \exp \left[- \left(1 + \frac{ \dot{\gamma} }{c} \right) \exp \left(- \frac{d}{ \dot{\gamma} } \right) \right], \mu_{\infty G} = 0.0035, n_{\infty} = 1.0, \Delta \mu = 0.025, \Delta n = 0.45, a = 50, b = 3, c = 50, \text{ and } d = 4$

doi:10.1371/journal.pone.0128178.t001

3.7 Cell-centered finite volumes

The same problems as before can be solved by cell-centered finite volumes. It is interesting to see how the imposition of boundary conditions is quite different.

We adopt a convention for the numeration of unknowns in structured quadrilateral finite volumes.

- We consider a “covering domain” $(x^-, x^+) \times (y^-, y^+)$. The mesh is provided by two arrays, X and Y , such that

$$x^- = X_1 < X_2 < \dots < X_{n_2+1} = x^+, y^- = Y_1 < Y_2 < \dots < Y_{n_2+1} = y^+. \quad (3.34)$$

- The **cell** with numbering (i, j) will be

$$V_{ij} = (X_i, X_{i+1}) \times (Y_j, Y_{j+1}). \quad (3.35)$$

- The **cell unknowns** have the same numbering as the corresponding cell, and it is located at the **nodes** given by the arrays \hat{X} and \hat{Y} :

$$W_{ij} \simeq w(\hat{X}_i, \hat{Y}_j) \quad (3.36)$$

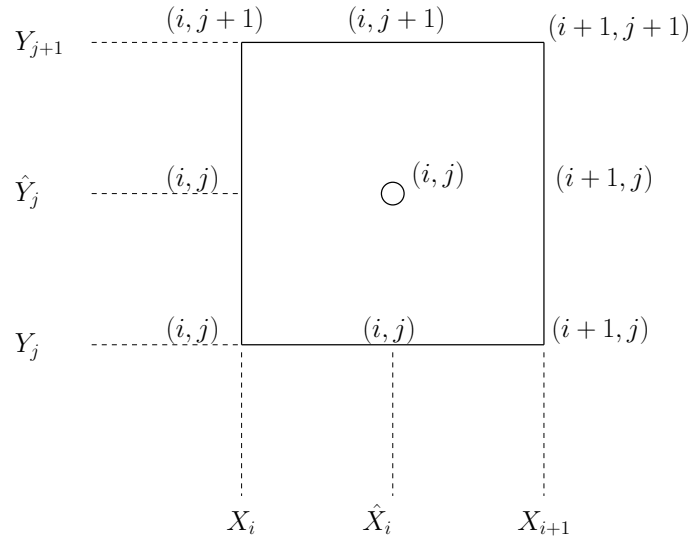
$$\hat{X}_i = \frac{1}{2}(X_i + X_{i+1}) \quad (3.37)$$

$$\hat{Y}_j = \frac{1}{2}(Y_j + Y_{j+1}) \quad (3.38)$$

$$\delta x_i = X_{i+1} - X_i \quad (3.39)$$

$$\delta y_j = Y_{j+1} - Y_j \quad (3.40)$$

- The face and vertex unknowns will be numbered as follows:



- All cells will have all unknowns, meaning that there will be:
 - $n_1 \times n_2$ cell unknowns.
 - $(n_1 + 1) \times (n_2 + 1)$ vertex unknowns.
 - $n_1 \times (n_2 + 1)$ horizontal face unknowns.
 - $(n_1 + 1) \times n_2$ vertical face unknowns.
- A **mask** will be a **cell variable** M_{ij} , with $1 \leq i \leq n_1$, $1 \leq j \leq n_2$, such that if $M_{ij} = 0$ we have a **fluid cell**. Other values of the mask will correspond to **walls**, which can have different boundary conditions depending on the value.

Exo. 3.4 Miniproject (electroosmotic pump 2): *Build a cell-centered code for the solution of the transient parallel flow of a viscous fluid with the following characteristics:*

- *The pressure gradient $\mathcal{G}(t)$ can be arbitrary.*
- *Time integration is performed with the method of lines (θ -method).*
- *$M_{ij} = 11, 12 \Rightarrow$ Smoluchowski condition, with m_{eo} given and $E_{\text{wall}}(t)$ programmable.*
- *Computes the flow rate $Q = \int_{\Omega} w \, d\Omega$ as a function of time.*

Then answer the same questions as in the previous miniproject.